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Ludwig

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(54) **SAFETY SWITCH FOR WELL OPERATIONS**

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E21B 17/00	(2006.01)
E21B 43/119	(2006.01)

(52) **U.S. Cl.**

CPC ***H01H 35/32*** (2013.01); ***E21B 17/00*** (2013.01); ***E21B 43/119*** (2013.01)

(58) **Field of Classification Search**

USPC 166/381, 113, 65.1

See application file for complete search history.

(56) **References Cited**

U.S. PATENT DOCUMENTS

2,897,308	A *	7/1959	Fergus	H01H 5/02 200/246
3,290,464	A *	12/1966	Finnegan	H01H 13/46 200/460
3,766,833	A *	10/1973	Kraemer	92/37
4,757,165	A *	7/1988	Marcoux	H01H 5/30 200/83 J
4,938,120	A	7/1990	Hafele et al.	
5,149,927	A *	9/1992	Stahly	H01H 35/343 200/83 J
2001/0052427	A1 *	12/2001	Eppink et al.	175/40
2002/0108747	A1 *	8/2002	Dietz et al.	166/66.7
2003/0051881	A1	3/2003	Vinegar et al.	

* cited by examiner

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(57) **ABSTRACT**

A pressure controlled safety switch comprises an electrical switch disposed in a cavity of a mandrel. A bellows assembly is operably engaged with the electrical switch. The bellows assembly is in fluid communication with a fluid surrounding the mandrel such that a pressure in the fluid no less than a predetermined pressure causes the bellows to activate the electrical switch, and a pressure in the fluid less than the predetermined pressure causes the bellows to deactivate the electrical switch.

15 Claims, 7 Drawing Sheets

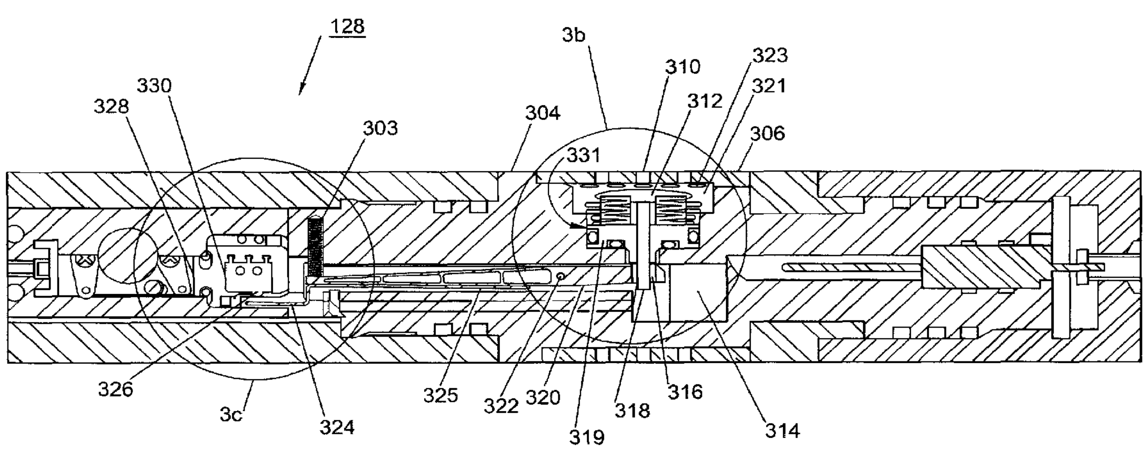

FIG. 1

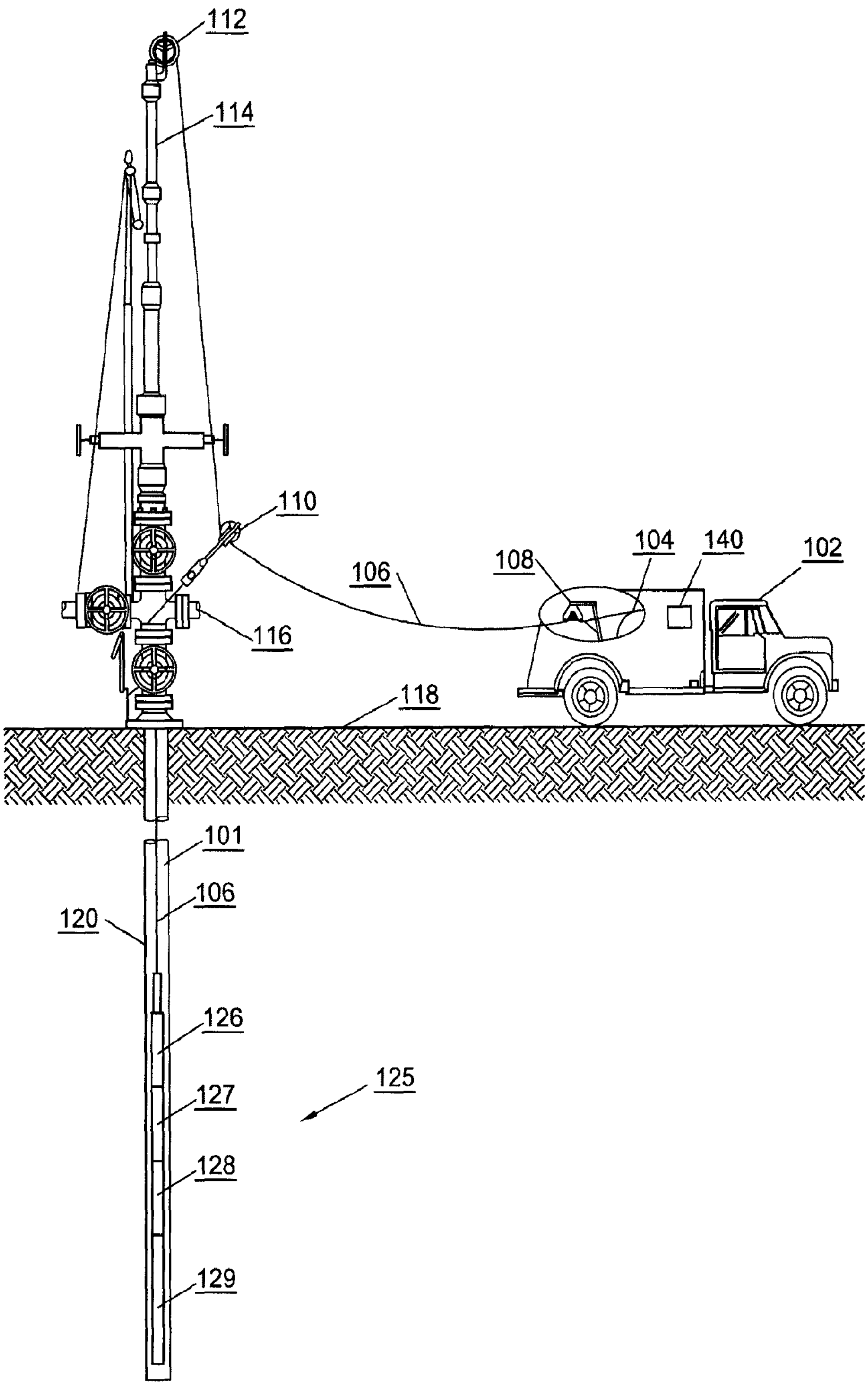
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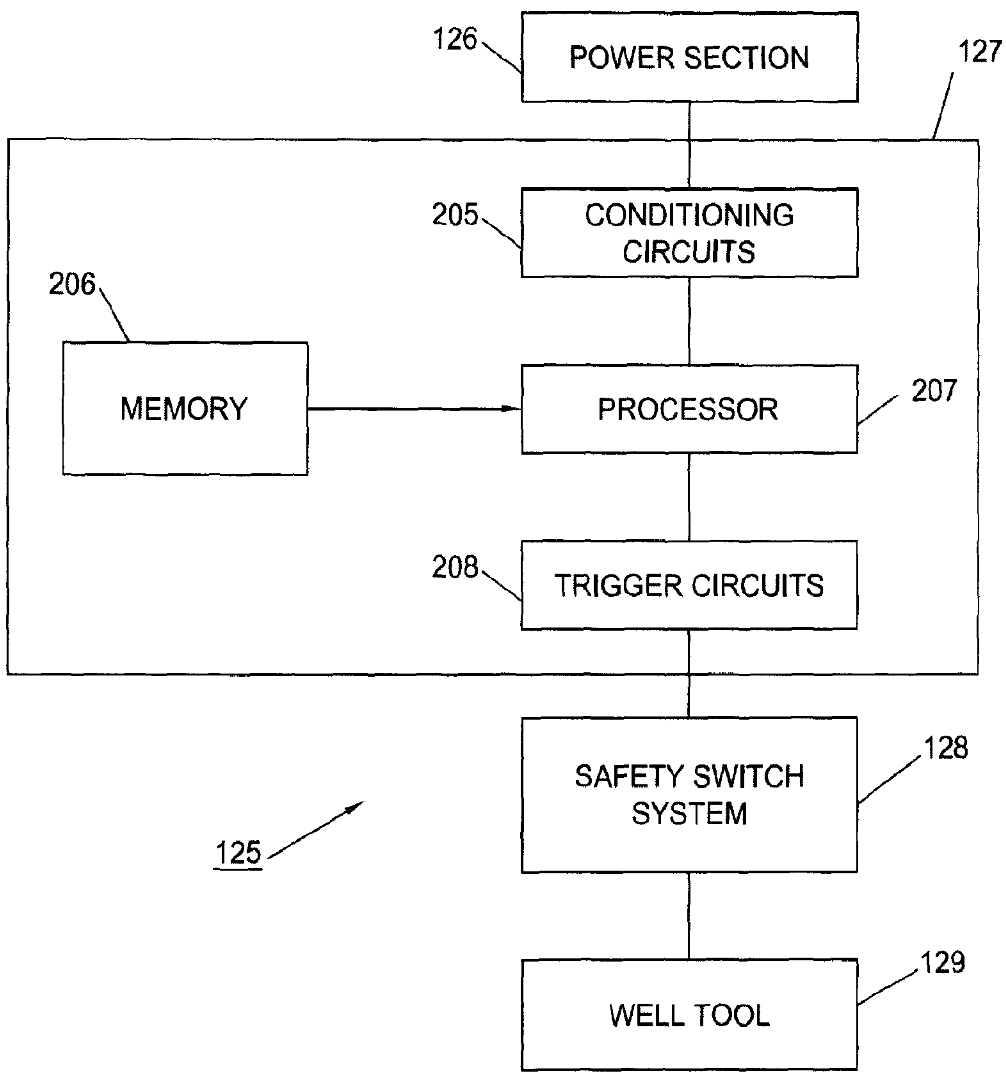
FIG. 2


126
POWER SECTION
127
205
CONDITIONING CIRCUITS
206
MEMORY
PROCESSOR
207
208
TRIGGER CIRCUITS
SAFETY SWITCH SYSTEM
128
125
WELL TOOL
129

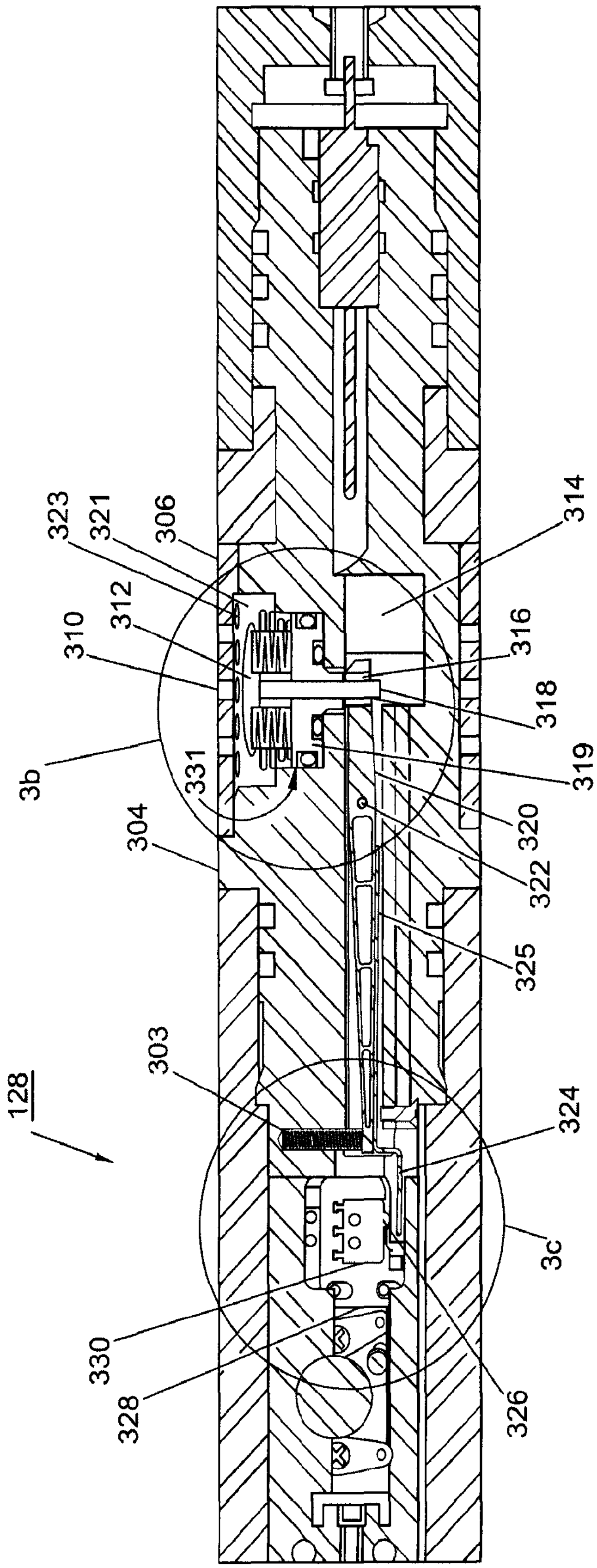

128
303
304
306
310
312
314
316
318
319
320
321
322
323
324
325
326
328
330
331
3b
3c

FIG. 3A

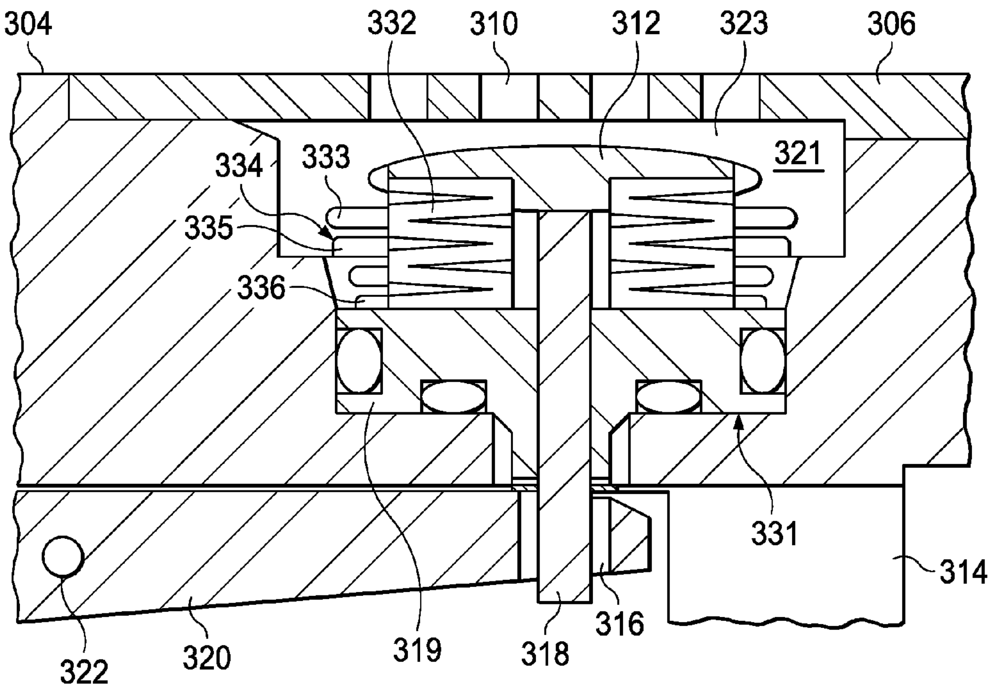

304
332
310
312
323
306
333
321
334
335
336
331
314
316
318
319
320
322

FIG. 3B

FIG. 3C

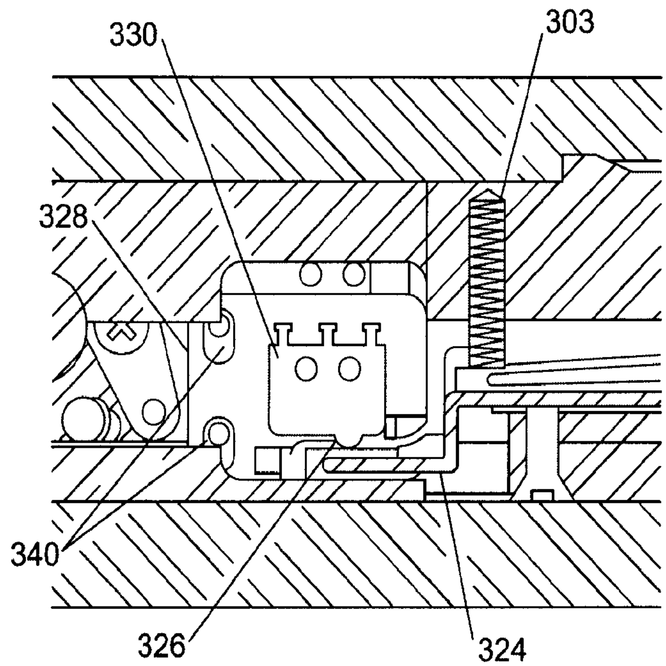


FIG. 4

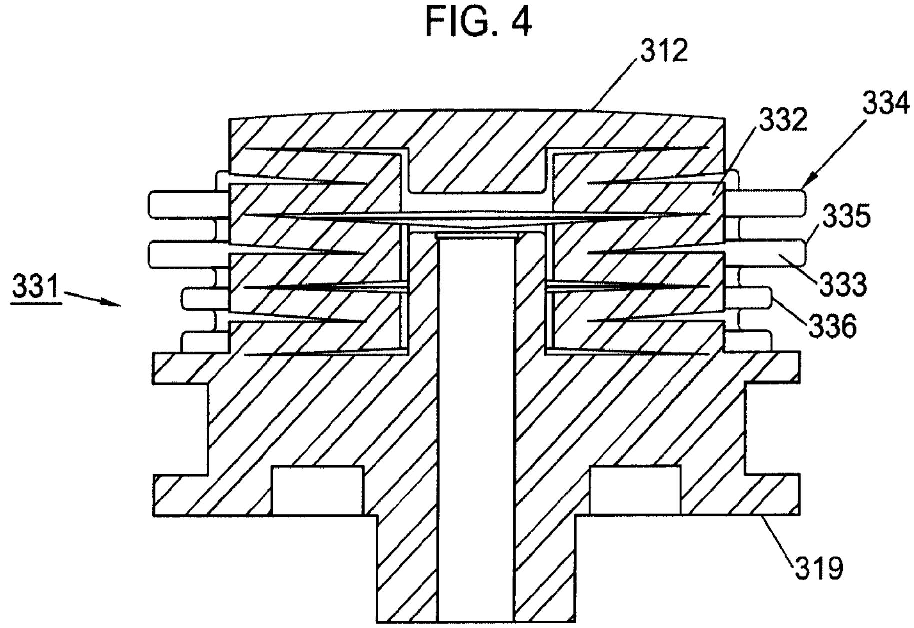


FIG. 5

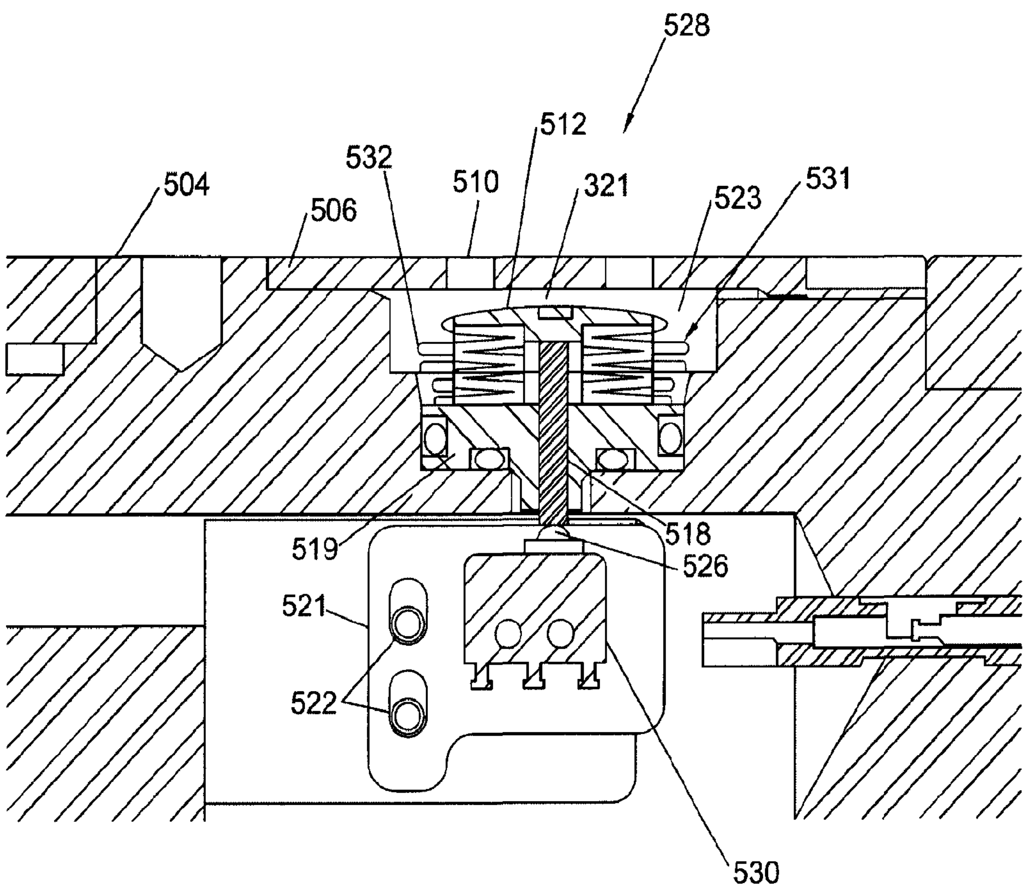
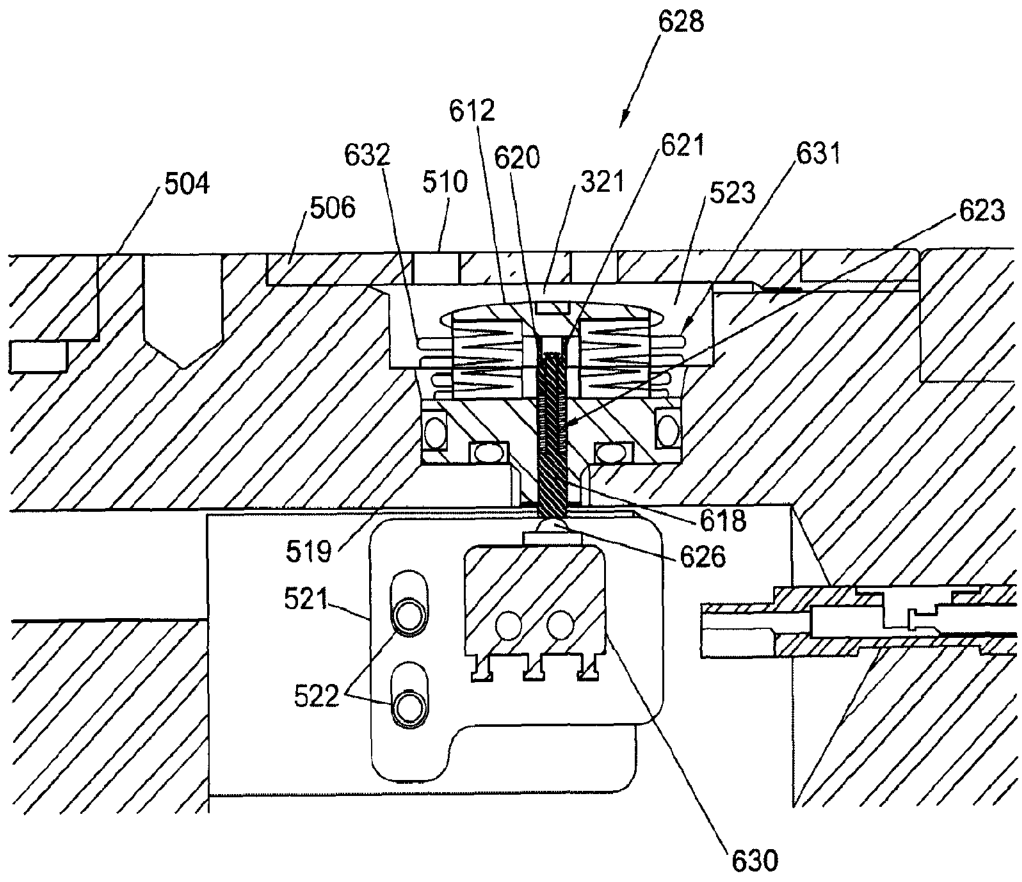


FIG. 6


628
612
632
620
621
631
510
321
523
504
506
623
618
519
626
521
522
630

SAFETY SWITCH FOR WELL OPERATIONS

BACKGROUND OF THE INVENTION

The present disclosure relates generally to the field of downhole tools for well operations.

As the oilfield industry moves to perform down hole operations as efficiently as possible, some operations formally done on wireline are being run on slickline, drill pipe, or other deployment means that do not contain an electrical conductor. Batteries may be used as a power source for these operations. For some types of operations, for example perforating, or running a neutron generator, accidental activation on, or near, the surface may cause injury to personnel and/or damage to equipment. Operations that were formally made safe by not applying power to the wireline are now being connected to a power source at the surface when, for example, the battery sub is activated or installed before descending down hole, resulting in the potential for surface activation if the device is mis-programmed, or has an electronics failure.

BRIEF DESCRIPTION OF THE DRAWINGS

A better understanding of the present invention can be obtained when the following detailed description of example embodiments are considered in conjunction with the following drawings, in which like elements are indicated by like reference indicators:

FIG. **1** shows an example of a rig-up for performing downhole well operations;

FIG. **2** shows a block diagram of one embodiment of a well tool string;

FIGS. **3**A-**3**C show cross sections of one example of a safety switch system;

FIG. **4** shows an example of a bellows assembly for use in a safety switch system;

FIG. **5** shows another example of a safety switch system; and

FIG. **6** shows yet another example of a safety switch system.

While the invention is susceptible to various modifications and alternative forms, specific embodiments thereof are shown by way of example in the drawings and will herein be described in detail. It should be understood, however, that the drawings and detailed description herein are not intended to limit the invention to the particular form disclosed, but on the contrary, the intention is to cover all modifications, equivalents and alternatives falling within the scope of the present invention as defined by the appended claims.

DETAILED DESCRIPTION

Described below are several illustrative embodiments of the present invention. They are meant as examples and not as limitations on the claims that follow.

FIG. **1** shows one example of a rig-up for performing down-hole well operations, also called well services, in a well bore **101** using a slickline **106**. As used herein, well operations may comprise logging, fishing, completions, perforating, workover operations, and combinations thereof. Well services truck **102** may contain a number of different features, for example, for this application, truck **102** contains drum **104**, which spools off slickline **106** through a combination measuring device/weight indicator **108**. Slickline **106** is rigged through lower sheave wheel **110** and upper sheave wheel **112**, and enters the well bore through pressure control equipment **114**, used to contain well bore pressure while

allowing slickline **106** to move freely in and out of the well bore. Slickline **106** enters the well bore at well head connection **116**, upon which pressure control equipment is connected. Below surface **118**, pipe or casing **120** may proceed to a bottom depth (not shown). Within casing **120** is well tool string **125**, connected to slickline **106**. Alternatively, well tool string **125** may extend into uncased sections of well bore **101**.

Combination measuring device and weight indicator **108** measures the motion of slickline **106** as it goes into and out of the well bore, and sends representative signals to a data handling system **140** disposed in truck **102** in order to provide the operator with accurate depth data. Additionally, in the example shown, combination measuring device and weight indicator **108** contains a cable tension measuring sensor and sends a signal into the logging compartment of truck **102**, indicating an increase in the tension on slickline **106**. Alternatively, any other technique, known in the art, may be used to determine line tension and tool depth.

As used herein, a slickline cable comprises a single strand strength member having a relatively smooth outer surface. While the slickline strength member may be metallic, it is not used to conduct electrical signals or power.

Alternatively, well tool **125**, may be run on drill pipe, coiled tubing, and any other suitable deployment technique known in the art. As used herein the term deploy is intended to mean extension and/or retrieval of a tool into the well.

In one example, well tool string **125** may comprise a power section **126** for supplying power to the downhole system. An electronics section **127** may be attached to power section **126**. A safety switch system **128** may be attached between electronics section **127** and a well tool **129**. Well tool **129** may comprise a logging tool, a completion tool, a fishing tool, a perforating tool, a workover tool, and combinations thereof.

FIG. **2** shows a block diagram of one embodiment of well tool string **125**. In one example, power section **126** may comprise batteries, for example lithium batteries. Alternatively, any other suitable battery type may be used. In another alternative, where flow is present in the wellbore, a downhole turbine generator may be used to extract power from the flowing fluid to generate suitable electrical power. As used herein, the term fluid is intended to comprise liquids, gases, liquid/solid mixtures, emulsions, and combinations thereof. The electronics section **127** may comprise suitable conditioning circuits **205** for powering the devices in electronics section **127**, any active devices in safety switch system **128**, and well tool **129**. Electronics section **127** may also comprise a processor **207** in data communication with a memory **206** that may have suitable programmed instructions stored therein for controlling operation of tool string **125**. Processor **207** may comprise one or more processors of a type known in the art. Processor **207** may act according to programmed instructions to activate trigger circuits **208** that activate well tool **125**.

In one embodiment, safety switch system **128** comprises a high reliability system that is capable of relatively low switching pressures and extremely high overpressures. The system may prevent a tool, for example a perforating tool, from activating until the system has descended to a depth that builds a predetermined activating pressure to activate the switch system to arm the tool. In addition, the pressure activated safety switch system disclosed herein will switch to a safe position, deactivating the tool, when the pressure decreases below a predetermined deactivating pressure to ensure the device is safe upon retrieval from the well. High reliability is achieved by using flexing elements with low friction and hysteresis, and by supporting the deformable parts when pressures greatly exceed the switching pressure.

A cross section of one example of safety switch system **128** is shown in FIGS. **3**A-**3**C and FIG. **4**. In one embodiment, safety switch system **128** comprises a bellows assembly **331** having an edge welded inner bellows **332** welded to a cap **312** at one end and a base **319** at an end opposite the cap **312**. The inner bellows is surrounded by an outer bellows **334**, and the space between inner bellows **332** and outer bellows **334** is filled with oil **333**. Alternatively, a rolling elastomeric bellows may be used as an outer bellows, for example a Bellofram brand bellows from Bellofram Corp, Newell, W Va. In yet another alternative embodiment, there may be no outer bellows such that formation fluid is in contact directly with inner bellows **332**.

Bellows assembly **331** is inserted in a cavity **323** in mandrel **304** and protected by cover **306**. Holes **310** in cover **306** allow wellbore fluid **321** to enter cavity **323**. The outside of outer bellows **334** and cap **312** are exposed to well bore fluid **321** in cavity **323**. The pressure of well bore fluid may be as high as 40,000 psi. The inside of mandrel **304** and inner bellows **332** is filled with a gas at substantially atmospheric pressure. The gas may comprise air, nitrogen, argon, other known inert gas, and combinations thereof. Pressure from the well bore fluid **321** acts to axially compress both bellows **334** and **332**. In one example, the outer bellows resistance to axial compression is considered negligible compared to the resistance of the inner bellows **332**. Once inner bellows **332** reaches its solid height, further increases in fluid pressure provide very little increase in stress, thus allowing bellows assembly **331** to handle a very high fluid pressure. In one example, outer bellows **334** has a small diameter section **336** and a large diameter section **335**. The geometry of outer bellows **334** and the geometry of inner bellows **332** determine the volume between the two. As inner bellows **332** is compressed, outer bellows **334** is compressed as well. The overall length gets shorter. The function of the two diameters of outer bellows **334** is to allow them to be sized so that as the bellows assembly is compressed, the length of the larger diameter section gets longer while the length of the small diameter section gets shorter at a rate greater than the overall deflection of the bellows assembly. This allows the outer bellows to compensate for volume changes of the fluid between the two bellows due to temperature and pressure across the full range of motion of the bellows assembly. The outer bellows **334** does not have to compress fully, and has very little pressure differential across it. This makes it much less subject to debris affecting its operation. Outer bellows **334** protects inner bellows **332** from accumulating well bore fluids and particulate matter between the convolutions of inner bellows **332** thereby eliminating failures due to these contaminations.

In one embodiment, the motion of the inner bellows **332** is transmitted to a lever arm **320** by a pin **318** through the center of inner bellows **332** and base **319**. Pin **318** engages lever arm **320** in slot **316**. Lever arm **320** pivots about pin **322** in internal cavity **314**. The other end of lever arm **320** is engaged with a compression spring **303**. The force of compression spring **303** holds pin **318** in compression, and resists movement of lever arm **320** and pin **318** due to shock and vibration. In addition, compression spring **303** may act to return lever arm **320** to the inactivated position as inner bellows **332** returns to its uncompressed position, as pressure is reduced. Lever arm **320** may also comprise a preloaded cantilever spring **324** attached to lever arm **320** at **325**. Cantilever spring **324** is formed to contact plunger pin **326** of switch **330**. The preload on cantilever spring **324** is sufficient to actuate switch **330** before cantilever spring **324** is deflected away from lever arm **320**. As inner bellows **332** moves through its range to a solid position, it causes lever arm **320** to move through its full range of motion. Cantilever spring **324** provides over travel of lever arm **320** past the actuation point of switch **330** without applying high forces to switch **330**. This overtravel allows the actuation point of electrical switch **330** to be set anywhere in the usable range of lever **320** travel and makes the switching pressure adjustable over a large percentage of inner bellows **320** travel. For example, the pressure switch assembly may be set such that the actuation of switch **330** is set to occur when inner bellows **330** is approximately at the mid-point of its travel. If the lever arm is directly in contact with the switch, additional external pressure will cause additional force on switch **330**, possibly damaging the switch. Cantilever spring **324** is substantially more flexible than lever arm **320** and imparts a much lower force to switch **330** during the additional travel of inner bellows **330** between the actuation point and the solid height of inner bellows **330**.

In one example, switch **330** may be a commercially available miniature switch, for example a Micro Switch brand switch from Honeywell, Inc. of Minneapolis, Minn. Switch **330** may be mounted to an adjustable carrier plate **328** that is controllably movable to provide the proper setting point for actuation of the switch at the appropriate travel point of inner bellows **332**. The position of plate **328** may be adjusted by turning an adjustment screw (not shown), and plate **328** may be locked in place with screws (not shown) installed in the slots **340** at the left side of carrier plate **328**. In one embodiment, switch **330** may have a spring return to return lever arm **320** to an inactivated position as pressure on the bellows **332** is reduced below the actuation pressure.

The deflection of inner bellows **332** due to pressure is linear over a substantial portion of its travel, but non linear effects may be present at both ends of travel. In one example, the switch position may be adjusted such that the actuation point is not near the inner bellows travel end points. It is intended that the switching system provide for operation over a switching range of 100 psi to 5000 psi and to operate in a 40000 psi downhole environment. In one embodiment, multiple bellows assemblies **331** may be used to cover different operating ranges over the desired switching range. Bellows assemblies **331** with different actuation pressure ratings can be interchanged to make the pressure switch system actuate over a wide selection of activation pressures. Each of the bellows assemblies will have the same mechanical stroke, but will reach full stroke at different maximum pressures.

In another embodiment, see FIG. **5**, safety switch system **528** comprises a bellows assembly **531** that has an edge welded bellows **532** welded to a cap **512** at one end and a base **519** at an end opposite the cap **512**. Alternatively, a double bellows assembly, as described previously, may be used. Bellows assembly **531** is inserted in a cavity **523** in mandrel **504** and protected by cover **506**. Holes **510** in cover **506** allow wellbore fluid **321** to enter cavity **523**. The outside of bellows **532** and cap **512** are exposed to well bore fluid **321** in cavity **523**. The pressure of well bore fluid may be as high as 40,000 psi. Pin **518** is attached to cap **512** and moves therewith. Pin **518** is in contact with the plunger pin **526** of switch **530**. The inside of mandrel **504** and bellows **532** is filled with a gas at substantially atmospheric pressure. As pressure in fluid **321** increases, bellows **532** collapses toward switch **530** in a predictable manner such that pin **518** depresses plunger pin **526** and actuates switch **530**. As shown in FIG. **5**, the set point for switch actuation must be close to the solid height of bellows **532** to prevent damage to switch **530** with additional fluid pressure. The set point may be adjusted by moving switch **530** up, or down, on mounting plate **521**, and locking switch **530** in place with screws **522**.

In yet another embodiment, see FIG. **6**, a bellows assembly **631** may be installed in mandrel **504**, described above. Bellows assembly **631** may comprise an edge welded bellows **632** welded to a cap **612** at one end and a base **519** at an end opposite the cap. Alternatively, a double bellows assembly, as described previously, may be used. The outside of bellows **632** and cap **612** are exposed to well bore fluid **321** in cavity **523**. The pressure of well bore fluid may be as high as 40,000 psi. The inside of mandrel **504** and bellows **532** is filled with a gas at substantially atmospheric pressure. A spring energized pin assembly **620** is attached to cap **612** and engages plunger pin **626** of switch **630**. Pin assembly **620** comprises a pin guide **621**, a compression spring **623** and a pin **618**. Pin guide **621** is in contact with cap **612**. Pin **618** extends slidably through pin guide **621** and may be retained on a top end by flaring the top of pin **618**. Alternatively, the top of pin **618** may be retained by a threaded fastener (not shown). The space between the bottom of pin guide **621** and the top shoulder of guide pin **618** captures compression spring **623**. In one example, this spring cavity may be shorter than the free length of compression spring **623**. This captures a pre-load on spring **623**. The pre-load on spring **623** may be designed to be greater than the switch actuation force. This causes the pin assembly **620** to act as a solid pin at, or below, the actuation force so that the pin exactly follows the motion of the bellows, and therefore gives a precise switch point. No further compression of the spring occurs until after the switch has been activated. After activation the spring can then be compressed and serves to limit the force applied to switch **623**. For example, as pressure in fluid **321** is increased, cap **612** is forced toward switch **630**, causing pin guide **621** to act against spring **623** that in turn causes pin **618** to impart a load to plunger pin **626** of switch **630**. Spring **623** may be preloaded, as described above, to impart sufficient force to actuate switch **630** at a desired motion point of bellows **632** as bellows **632** collapse. This allows the activation and deactivation of safety switch **628** at a predetermined fluid pressure. In addition, spring **623** may be designed using techniques known in the art to limit the load imparted to switch **630** to an allowable load for the selected switch. Spring **623** may comprise at least one of, a coil spring, a wave spring, and a disc spring.

In one operational example, a perforating operation may be desired at a predetermined location in the wellbore. Knowing the location, the downhole pressure may be estimated. In addition other operational situations may be considered. For example, on an offshore well, it would be prudent to set the switch activating and deactivating pressure to a level that ensures that the system is not armed until the perforating gun is below the sea bed level to prevent a misfire of the gun in the marine riser section. Once the appropriate activating and deactivating pressure is determined, the appropriate bellows assembly may be selected and installed in the switch mandrel. This may be done in the shop before the tool is sent to the rig, or alternatively, may be done in the field. As the tool is lowered to the preselected depth, the downhole pressure activates the switch, allowing operation of the well tool. During retrieval of the tool the switch is deactivated at the desired operating pressure to prevent well tool operation during the remainder of the retrieval cycle. For example, a misfiring perforating gun may be deactivated to prevent accidental firing near, or at, the surface. Other well and logging tools, for example a logging neutron generator known in the art, may use the safety switch described herein to ensure personnel and operational safety.

Numerous variations and modifications will become apparent to those skilled in the art. It is intended that the following claims be interpreted to embrace all such variations and modifications.

The invention claimed is:

1. A pressure controlled safety switch comprising:

an electrical switch disposed in a cavity of a mandrel; and

a bellows assembly operably engaged with the electrical switch, the bellows assembly in fluid communication with a fluid surrounding the mandrel such that a pressure in the fluid no less than a predetermined pressure causes the bellows to activate the electrical switch, and a pressure in the fluid less than the predetermined pressure causes the bellows to deactivate the electrical switch; a pivoted lever arm disposed between and operatively coupled to the electrical switch and the bellows assembly such that movement of the bellows assembly causes the lever arm to activate and deactivate the electrical switch;

wherein the pivoted lever arm further comprises a cantilever spring attached to the pivoted lever arm, and an end of the cantilever spring engages the electrical switch.

2. The pressure controlled switch of claim **1** wherein the bellows assembly comprises an inner bellows and an outer bellows.

3. The pressure controlled switch of claim **2**, wherein the outer bellows comprises at least one of a metallic bellows and an elastomer bellows.

4. The pressure controlled switch of claim **1** wherein the bellows assembly comprises a plurality of bellows assemblies wherein each of the bellows assemblies is sized to cover a separate predetermined pressure range.

5. The pressure switch of claim **4** wherein the electrical switch is located such that actuation occurs in the range of about 20% to about 80% of a total travel of the bellows assembly.

6. The pressure controlled switch of claim **1** wherein the predetermined pressure comprises a range of about 100 psi to about 5000 psi, and the pressure in the fluid comprises a range of about 100 psi to about 40000 psi.

7. The pressure controlled switch of claim **1** wherein the bellows assembly further comprises a spring energized pin engaged with the electrical switch.

8. A well tool string comprising:

a power section;

a well tool; and

a pressure controlled safety switch operatively coupled to both the power section and the well tool to operatively couple the well tool and the power section when a wellbore pressure is no less than a predetermined pressure and to operatively uncouple the well tool and the power section when the wellbore pressure is less than the predetermined pressure; wherein the safety switch comprises:

an electrical switch disposed in a cavity of a mandrel in the well tool string;

a bellows assembly operably engaged with the electrical switch, the bellows assembly in fluid communication with a fluid surrounding the mandrel such that a wellbore pressure no less than a predetermined pressure causes the bellows to activate the electrical switch, and a wellbore pressure less than the predetermined pressure causes the bellows to deactivate the electrical switch; and

a pivoted lever arm disposed between and operatively coupled to the electrical switch and the bellows assembly such that movement of the bellows assembly causes the lever arm to activate and deactivate the electrical switch;

wherein the pivoted lever arm further comprises a cantilever spring attached to the pivoted lever arm, and an end of the cantilever spring engages the electrical switch.

9. The well tool string of claim **8** wherein the electrical switch is located such that actuation occurs in the range of about 20% to about 80% of a total travel of the bellows assembly.

10. The well tool string of claim **8** wherein the predetermined pressure comprises a range of about 100 psi to about 5000 psi, and the pressure in the fluid comprises a range of about 100 psi to about 40000 psi.

11. The well tool string of claim **8** wherein the bellows assembly further comprises a spring energized pin engaged with the electrical switch.

12. A method for controlling activation of a well tool comprising:

selecting a bellows assembly for operation at a predetermined pressure;

installing the bellows assembly in a mandrel in a tool string;

operatively coupling the bellows assembly to an electrical switch located in the mandrel;

exposing the bellows assembly to a first pressure in a fluid surrounding the well tool no less than the predetermined pressure causing the bellows to activate the electrical switch;

wherein the coupling of the bellows assembly and the electrical switch further comprises a pivoted lever arm disposed between and operatively coupled to the electrical switch and the bellows assembly such that movement of the bellows assembly causes the lever arm to activate the electrical switch;

wherein the pivoted lever arm further comprises a cantilever spring attached to the pivoted lever arm, and an end of the cantilever spring engages the electrical switch.

13. The method of claim **12** further comprising exposing the bellows assembly to a second pressure in the fluid surrounding the well tool less than the predetermined pressure causing the bellows assembly to deactivate the electrical switch.

14. The method of claim **12** further comprising selecting the bellows assembly such that actuation of the electrical switch occurs in the range of about 20% to about 80% of a total travel of the bellows assembly.

15. The method of claim **12** wherein operatively coupling the bellows assembly to an electrical switch located in the mandrel comprises operatively engaging a first end of the pivoted lever arm to the electrical switch and operatively engaging a second end of the pivoted lever arm to the bellows assembly.

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